



Research Challenge 3

**Advance performance of energy
transformation technologies**

Ulster University

Professor NJ Hewitt, Dr Minjun Huang

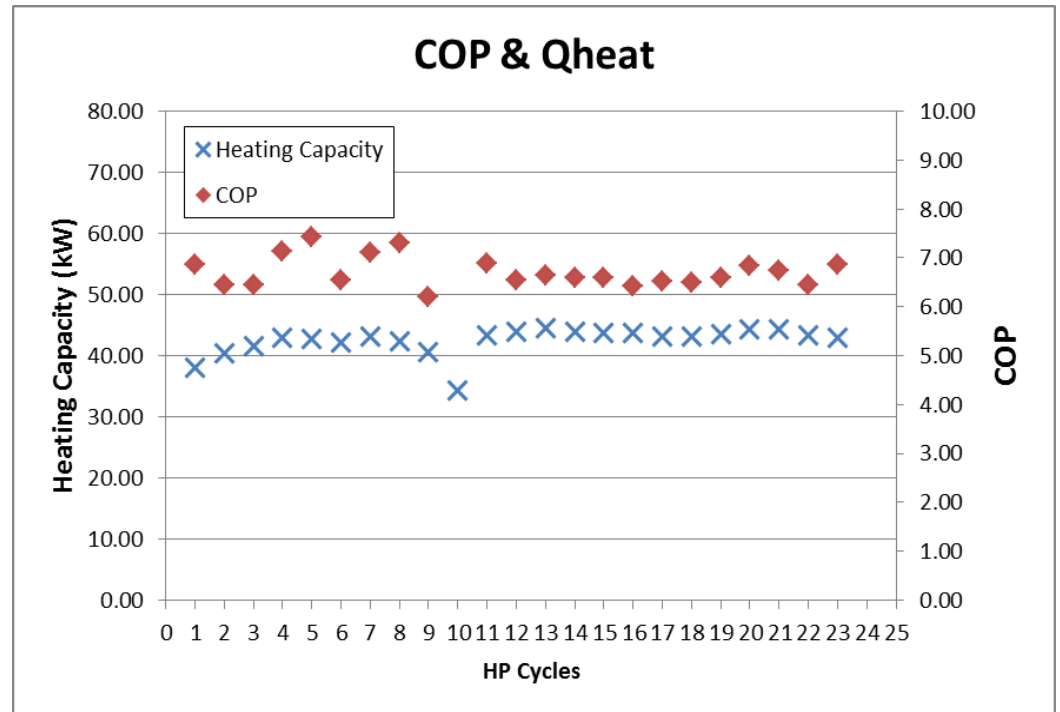
Dr Nik Shah, Dr Chris Wilson

Dr Donal Cotter

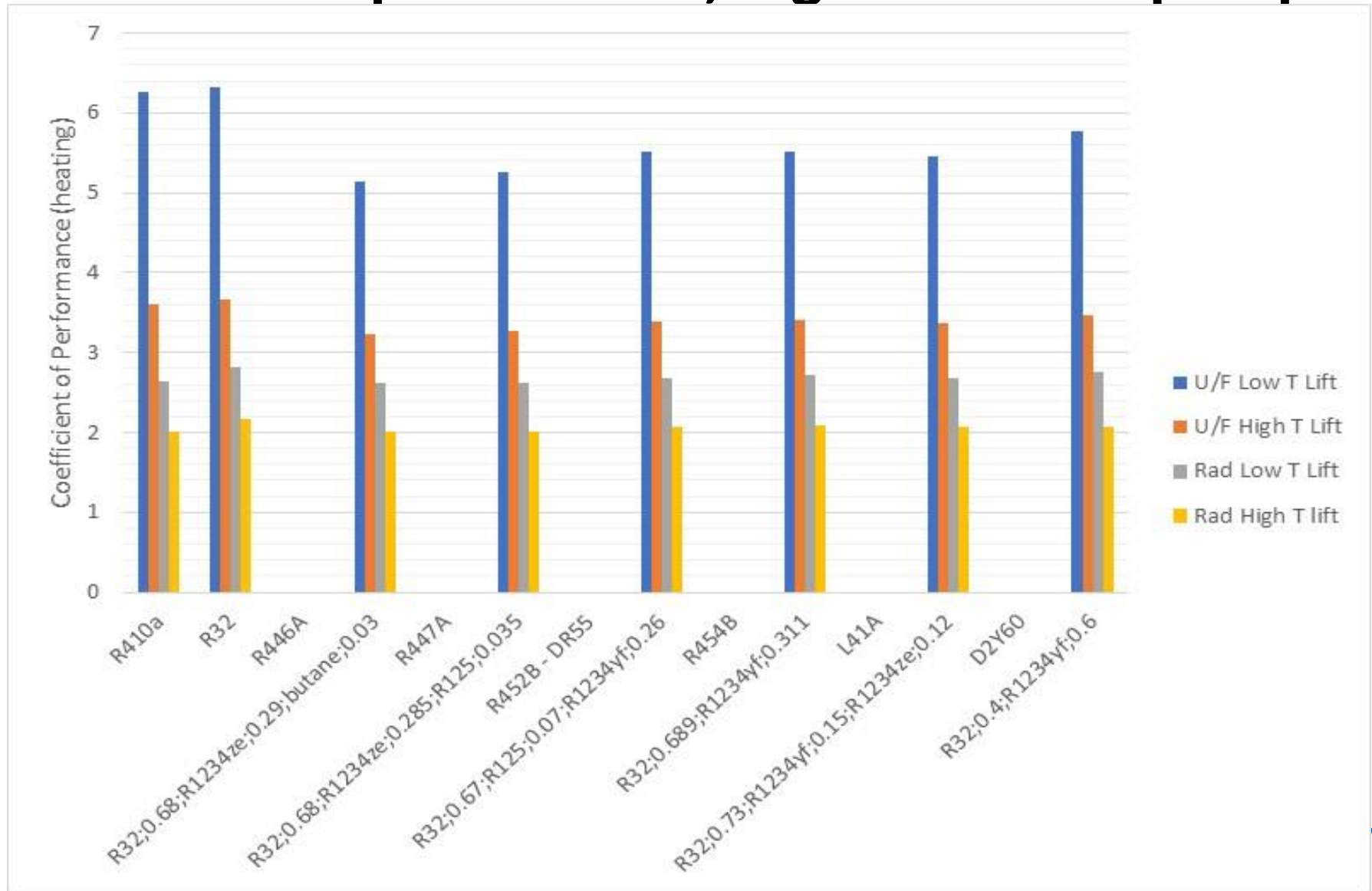


3.1 Low temperature lift, high COP heat pump

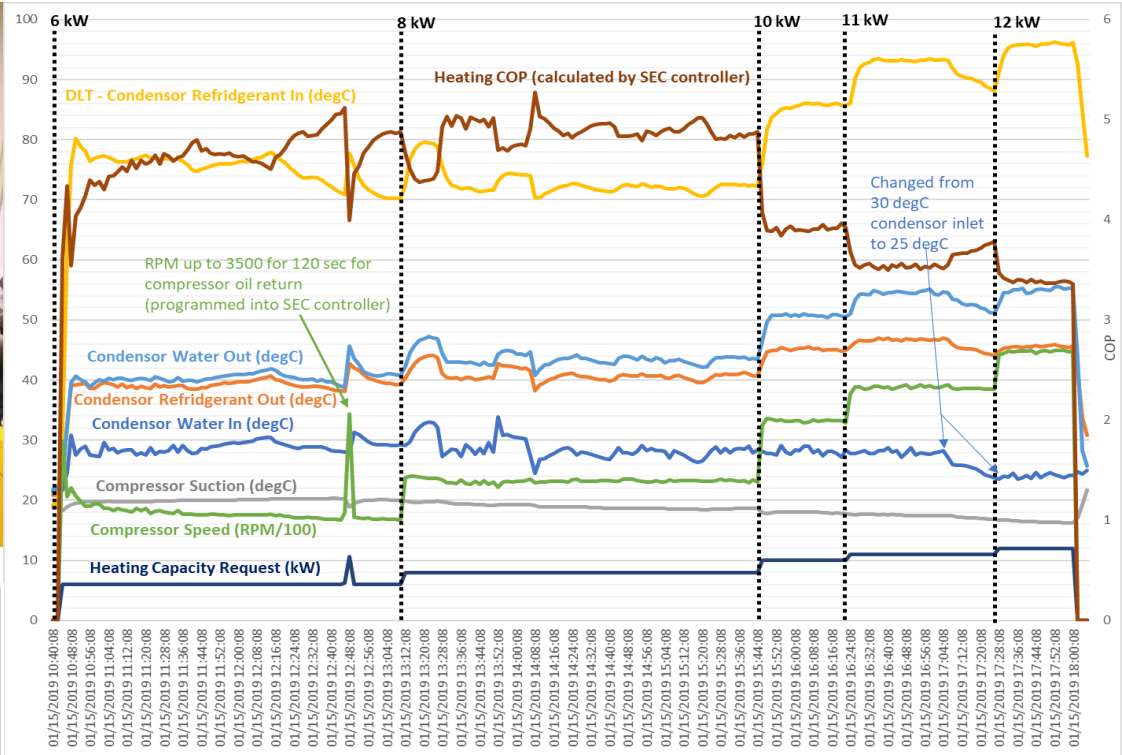
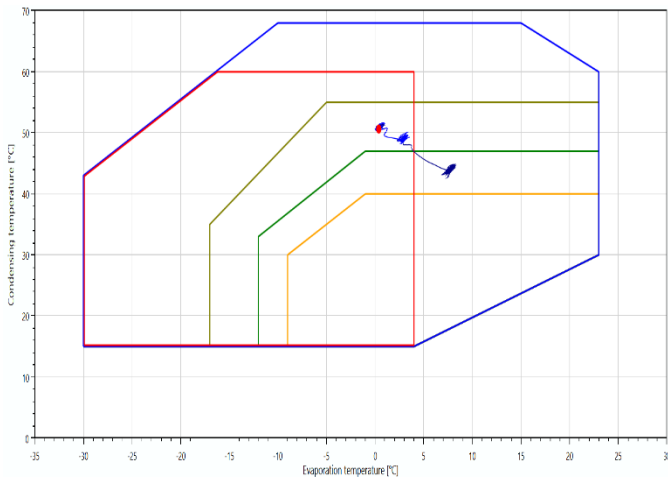
HP operation	November
Average Tamb (Low/High)	1°C/5°C
No of Cycles	24
Average operation time/cycle	36 minutes
Superheat setting	8°-12°K
Subcooling	11.3°K
STES Average temperature	34.8°C
Max. HP Water Outlet	52°C
Average Heating Capacity	43kW
Average COP	7.43



3.1 Low temperature lift, high COP heat pump



3.2 Heat Pump for Demand Side Management



3.2 Heat Pump for Demand Side Management

- Test Chamber – Humidity and ambient temperature control



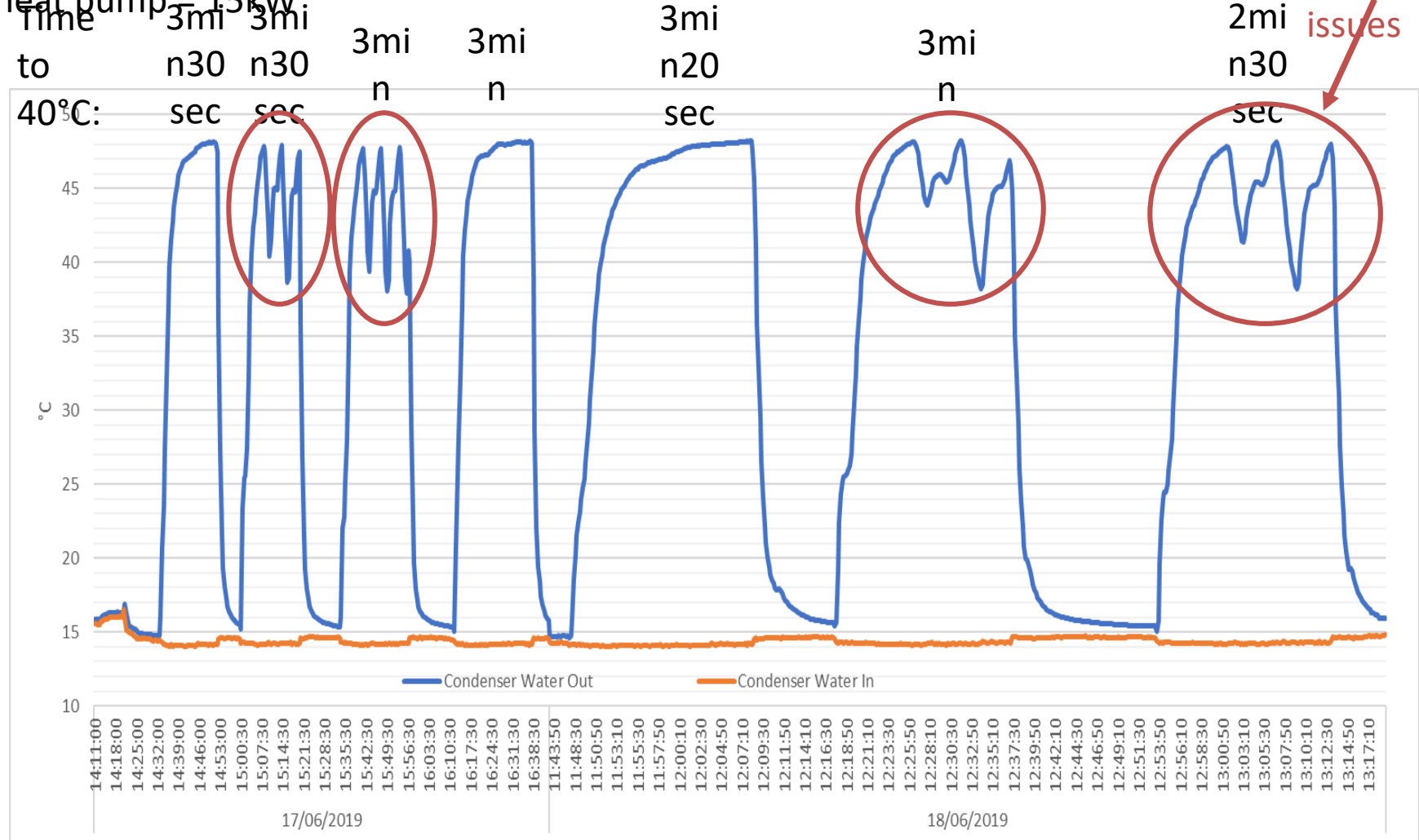
- HP and balancing rig moved to test chamber and plumbed in
- Sensor set-up and wiring completed for HP and test chamber
- PID control of chamber humidity reinstated



3.2 Heat Pump for Demand Side Management

Circles highlight HP control issues

Chamber = 15°C/RH90%, water flow rate = 6l/min, heat demand requested from heat pump = 15kW



Logging resolution every 30 seconds

Logging resolution every 10 seconds – 30seconds too short to see difference

3.2 Heat Pump for Demand Side Management

Modbus monitoring ver. 28.2.0.0



Modbus address

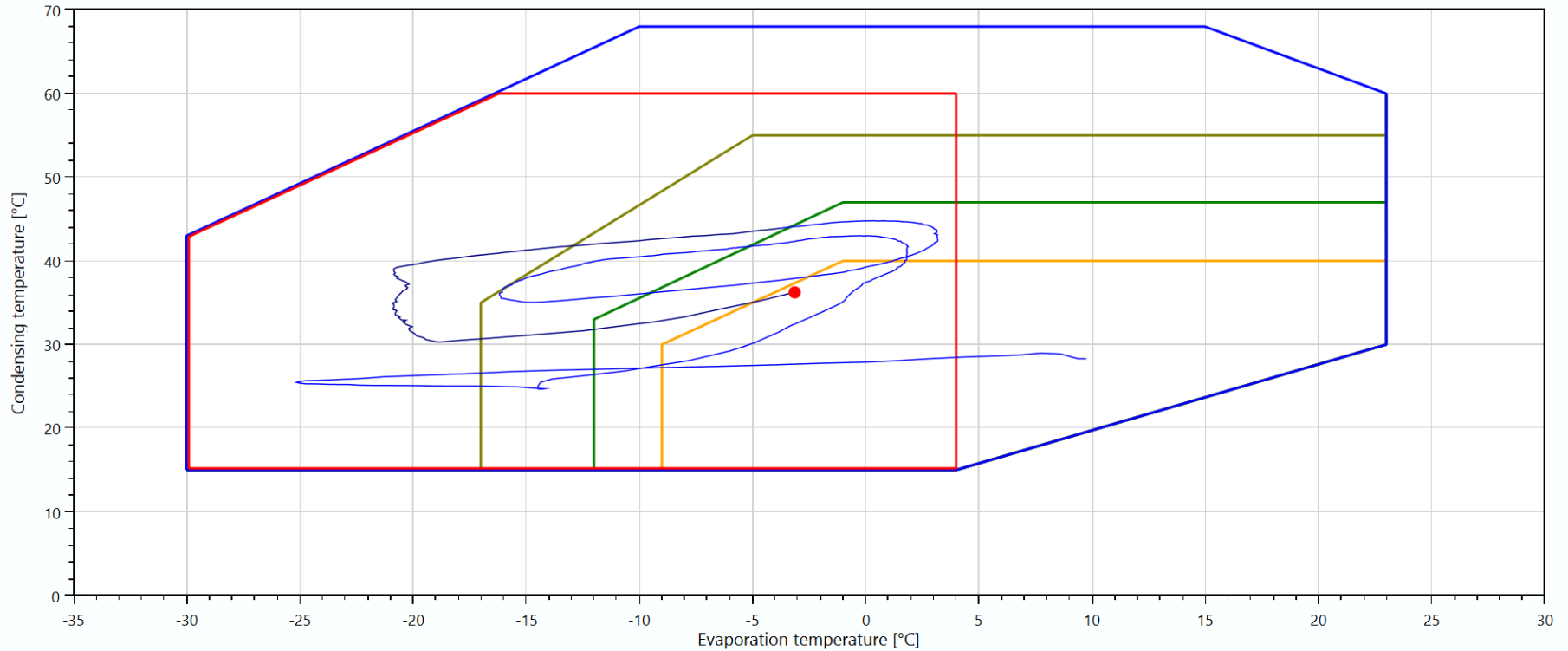
Get configuration from file

Get and write configuration from file

Write configuration to file

Startup | Monitoring | Configuration | Write | Alarm logging | Graphics | Envelope 1

Operating point (press F1 for help)



MultipleRead(03) - 00 - NoError

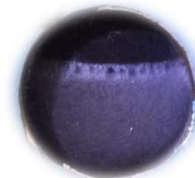
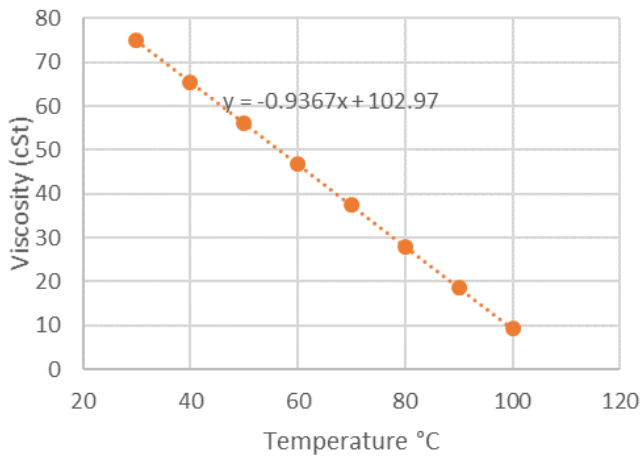
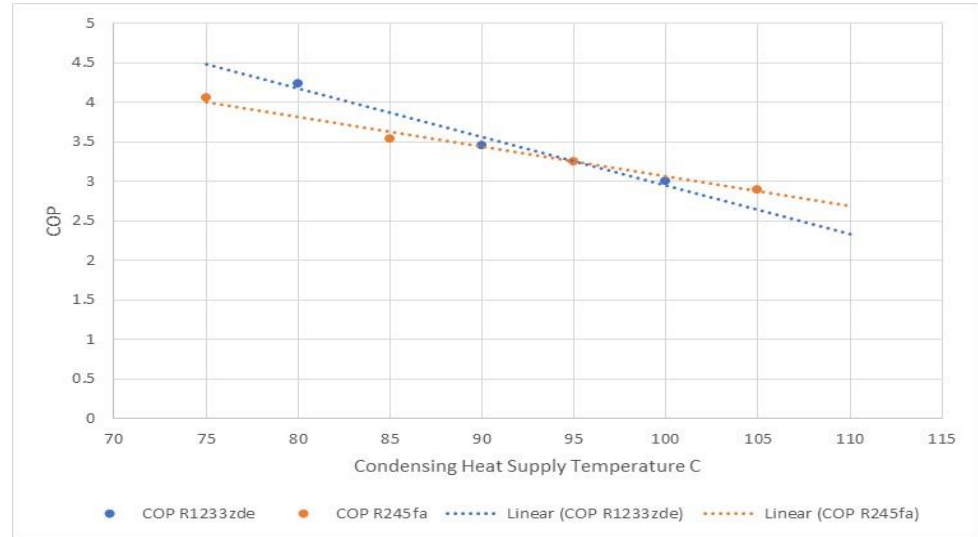
No COM failure

Options

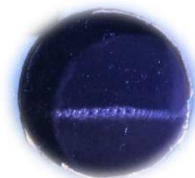
About



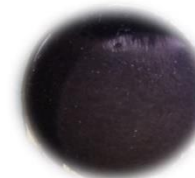
3.3 High temperature heat pumps



2:00pm



2:15pm



3:15pm

3.3 High temperature heat pumps



3.3 High temperature heat pumps

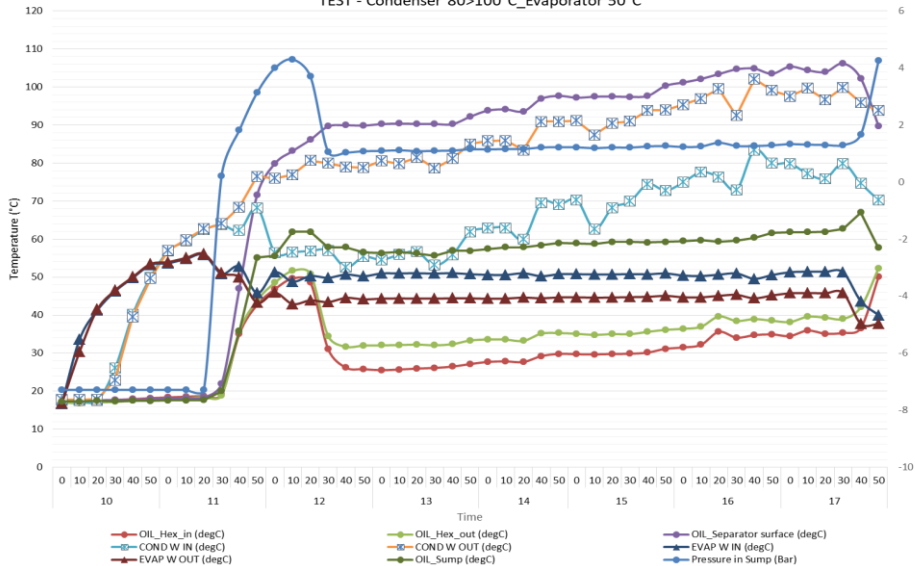
- **Expansion valve – failure $>100^{\circ}C$.**
 - Upon investigation the stepper motor had failed.
- **Compressor oil shaft seals failed – (Between compressor and motor)**
 - High oil temperature may be responsible.
 - Some wear on internal parts evident on sump plug.
- **Oil sump temperature**
 - Increases above $>90^{\circ}C$ Condensing temperatures.
- **Start-up Issues**
 - Liquid refrigerant in suction line.



3.3 High temperature heat pumps

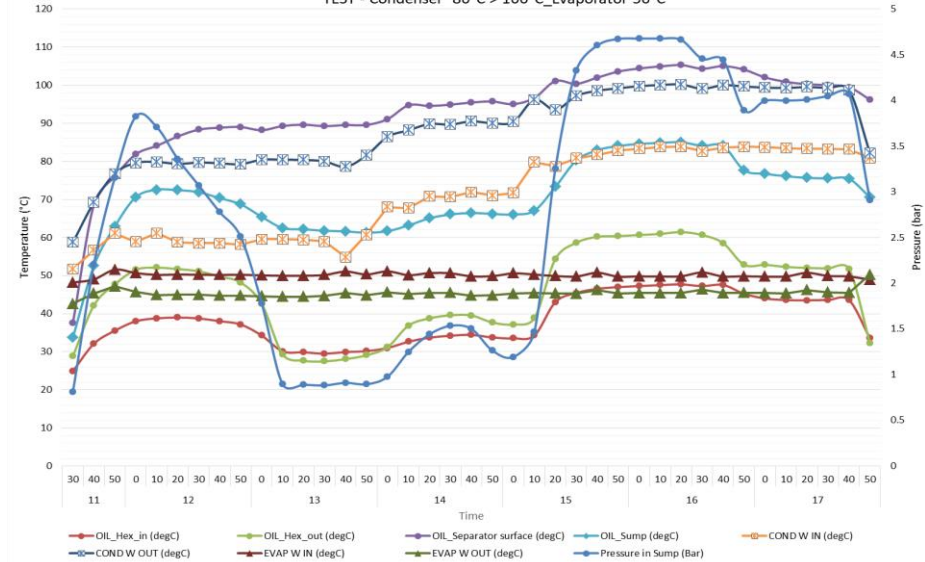
R245fa

TEST - Condenser 80>100°C_Evaporator 50°C



R1233zd

TEST - Condenser 80°C > 100°C_Evaporator 50°C



20cST is the target at operation points.

Suggestions: Fuchs RENISO TRITON SEZ320 (POE).

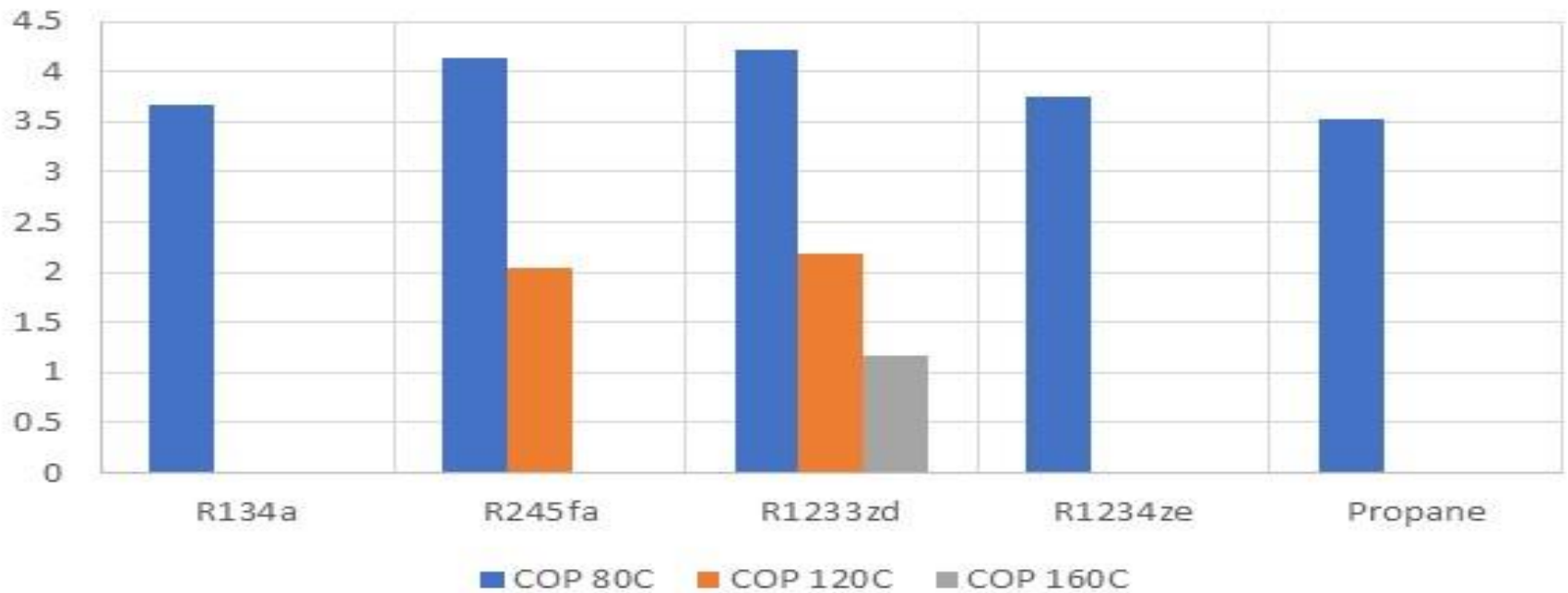
POE suitable with HCFO and miscible.

Other potential oil:

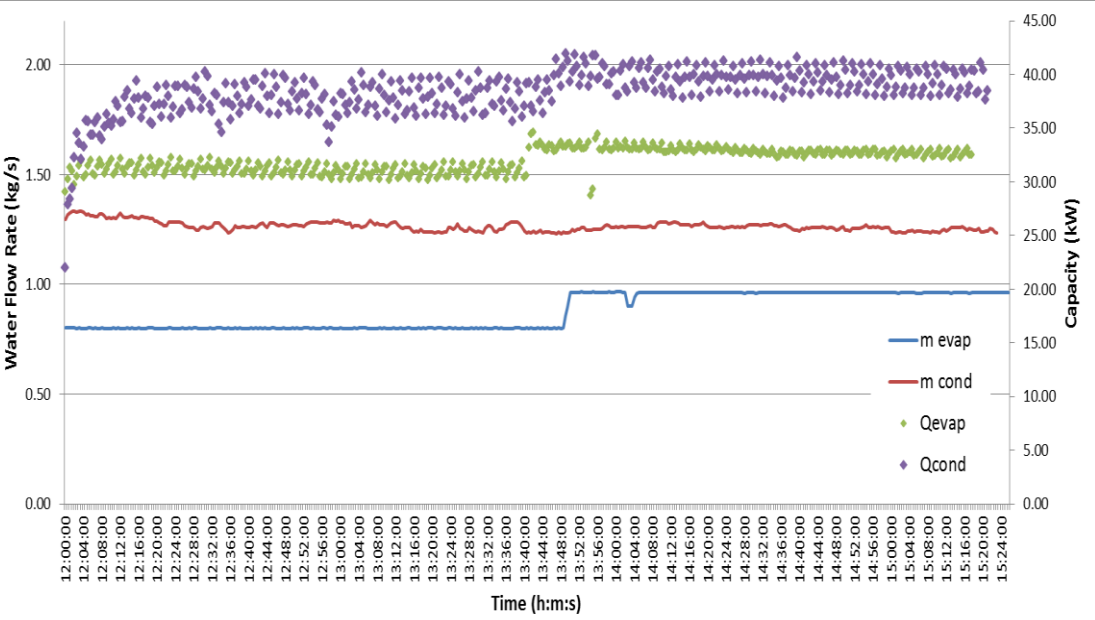
PAG but not tested for long terms performance, miscibility, acidity, corrosivity etc

Replacing R245fa

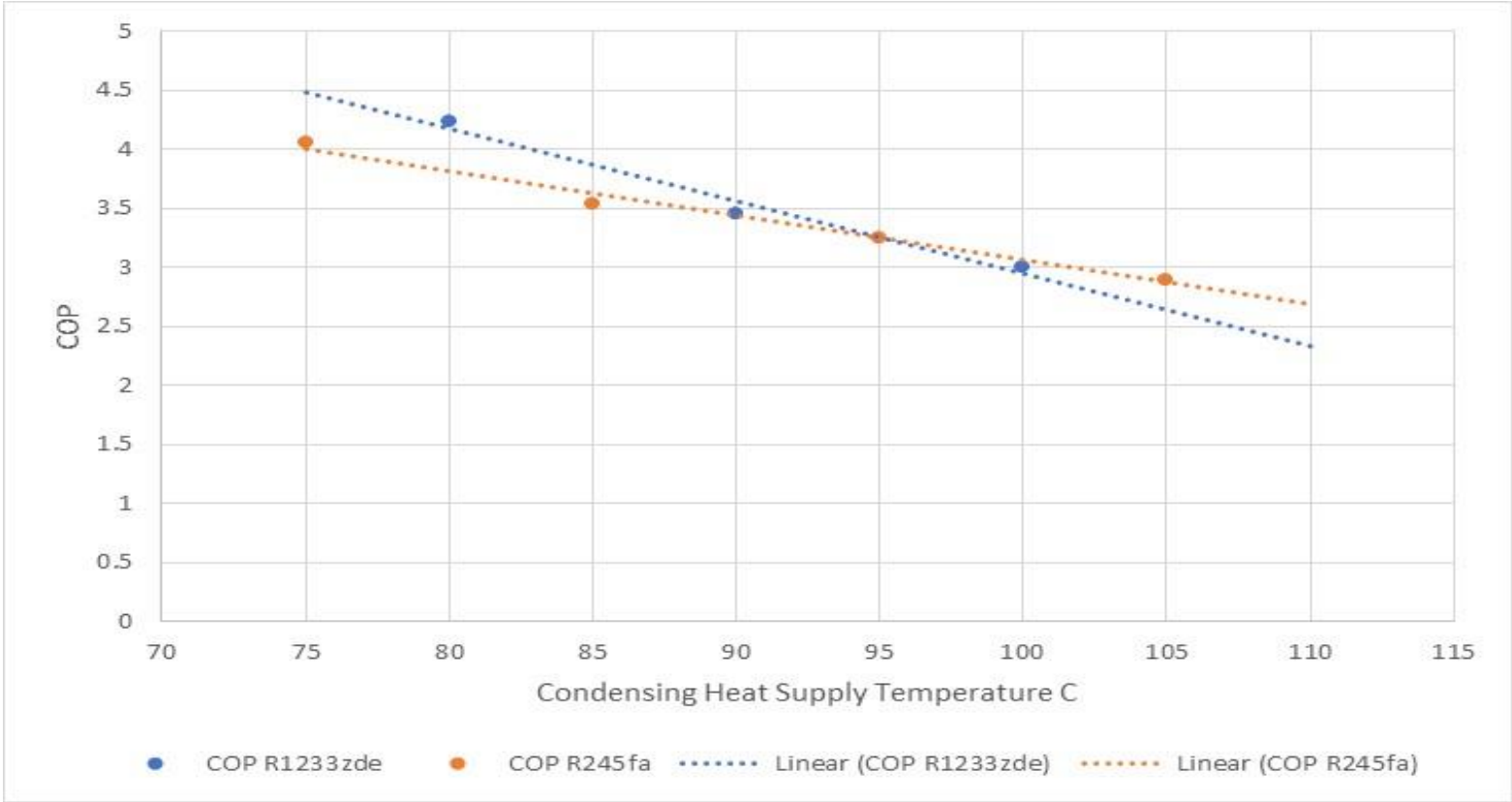
COP



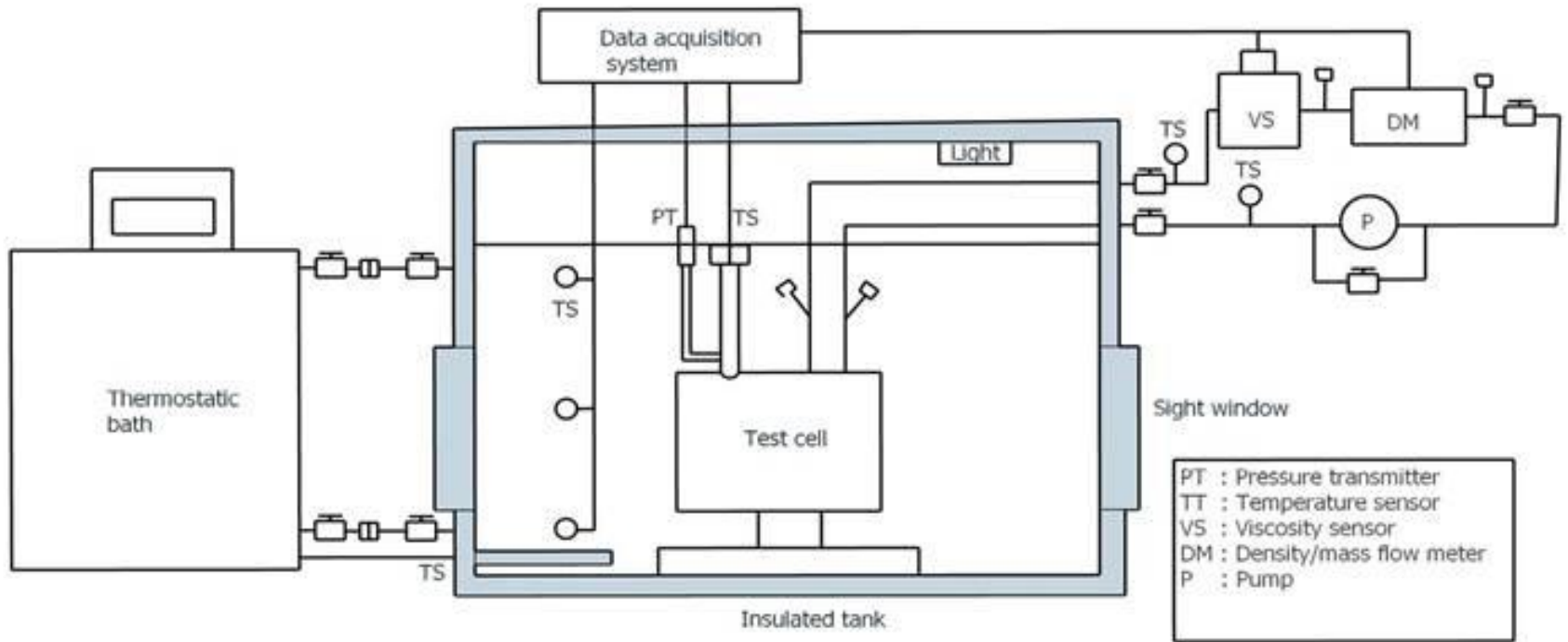
Replacing R245fa - Baseline



R1233zd(e) Performance



3.3 High temperature heat pumps



Schematic of the test equipment

Viscometer Test Rig

3.3 High temperature heat pumps

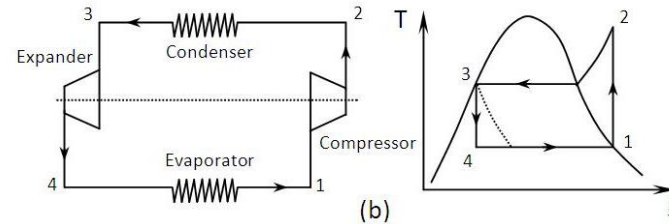


Viscometer Test Rig

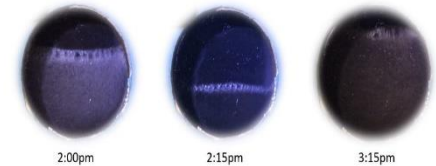


3.4 Combined heat pump/ORC

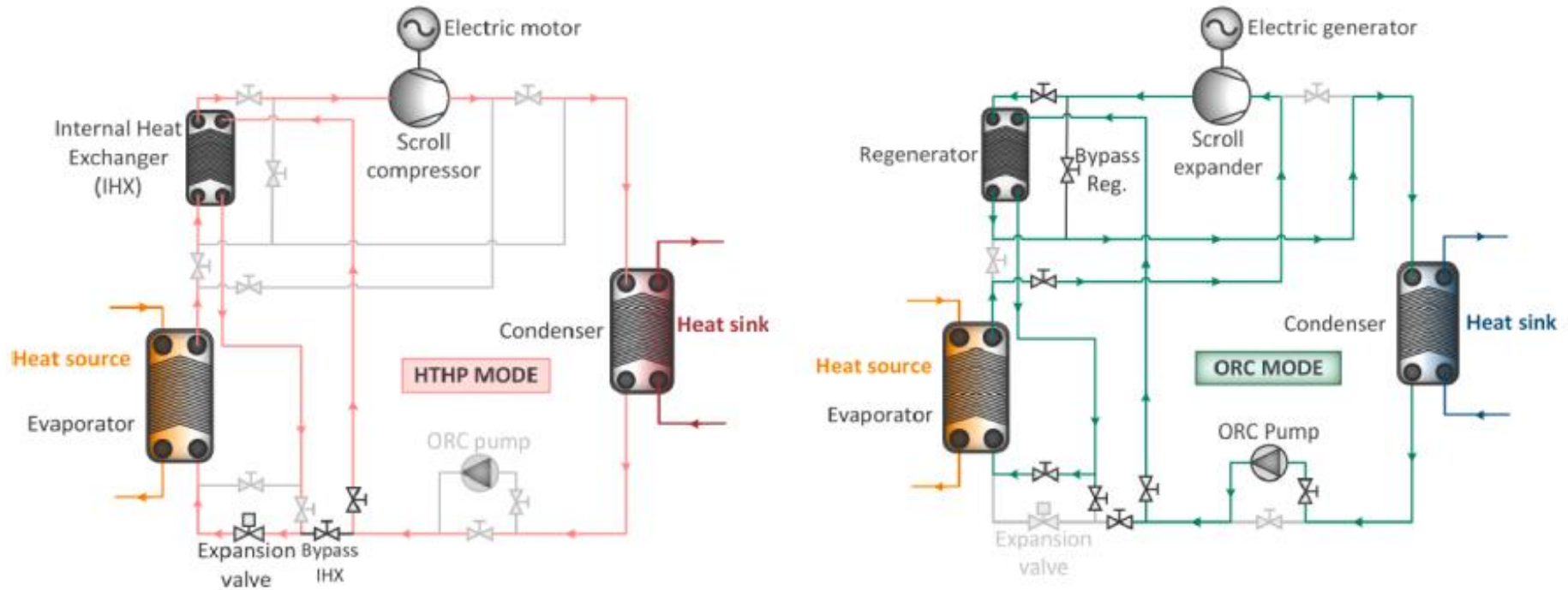
- Industrial applications may want to generate power from excess heat in two systems



- Stage 1 – Expander
- Stage 2 – Compressor as a Pump?
- Stage 3 – ORC/Heat Pump



3.4 Combined heat pump/ORC

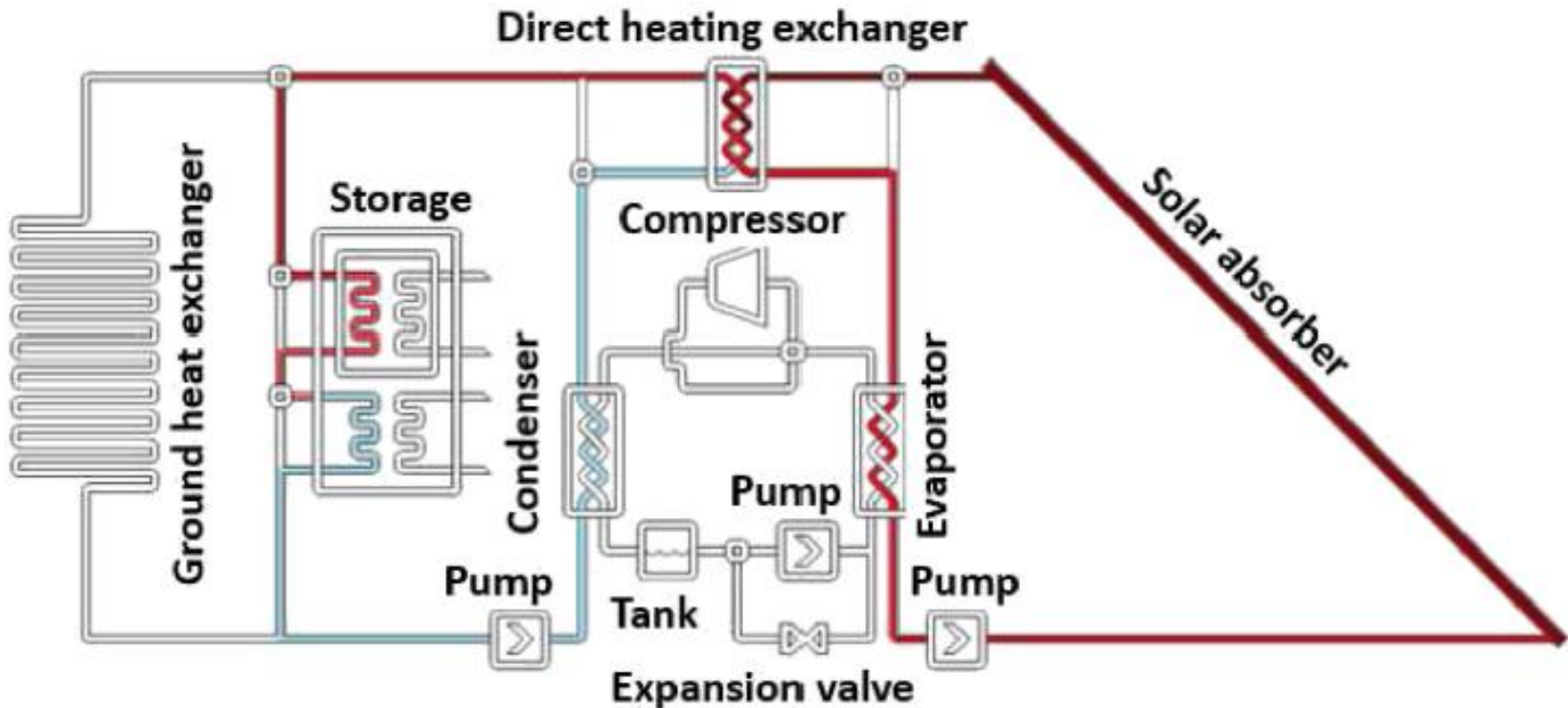


Multi-objective optimization of a novel reversible High-Temperature Heat Pump-Organic Rankine Cycle (HTHP-ORC) for industrial low-grade waste heat recovery

Energy Conversion and Management, Volume 197, 1 October 2019, Article 111908

Carlos Mateu-Royo, Adrián Mota-Babiloni, Joaquín Navarro-Esbrí, Bernardo Peris, Marta Amat-Albuixech

3.4 Combined heat pump/ORC



Experimental investigation of a reversible heat pump/organic Rankine cycle unit designed to be coupled with a passive house to get a Net Zero Energy Building

International Journal of Refrigeration, Volume 54, June 2015, Pages 190-203

Olivier Dumont, Sylvain Quoilin, Vincent Lemort

Thank you.